



WELCOME
TO Adda247

*"There is
nothing
impossible to
they who will
try."*

ISRO | BHEL | DRDO & OTHER PSUs



PRODUCTION

METROLOGY

MOST EXPECTED QUESTIONS

Live @ 11:30Am

PART-1



Gaurav sir



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GATE 2023 RESULT



Congratulations
FROM ADDA 247 FAMILY

AIR 03 ME KUSHAGRA DUTT	AIR 05 PI HARSHIT KUMAR	AIR 07 ME RUSHI PRADIPKUMAR KARIYA	AIR 11 CE VINEET JAIN	AIR 30 CE DITIK BANSAL	AIR 36 ECE SURIT KUMAR
AIR 64 CE UTKARSH MISHRA	AIR 71 EE SONESH SANJAY PAWAR	AIR 76 CE DIPANKAR DAS	AIR 87 EC SURAJIT RABI DAS	AIR 91 EE RISHABH GUPTA	AIR 111 ES ANIL GUPTA
AIR 130 EE SAURAV PATEL	AIR 136 CE RUPESH SACHDEVA	AIR 200 ECE WASIUZZAMA	AIR 212 IN WASIUZZAMA	AIR 217 ME VISHAL KUMAR	AIR 219 ME RITESH KUMAR
AIR 258 EE MANAV	AIR 348 EE AMAN NAMDEV	AIR 392 EE CAURAV MAHAJAN	AIR 403 EC MOHAN KUMAR SINGH	AIR 567 EE SHANKAR JHA	AIR 571 ME VIJENDER MEENA

You Tube Classes Schedule



MECHANICAL ENGINEERING

EXAM TARGET	SUBJECT	TIME	FACULTY
ALL PSUs	ENGINEERING MATHS	10:00 AM	ANANT SIR
ALL PSUs	PRODUCTION	11:30 AM	GAURAV SIR
ALL PSUs	THERMODYNAMICS	3:00 PM	KANISTH SIR
GATE 2024-25	HMT	4:30 PM	YOGESH SIR
GATE 2024-25	SOM	9:00 PM	MUKESH SIR

FREE APP CLASS SCHEDULE



MECHANICAL ENGINEERING



HMT	MONDAY Live @11AM	YOGESH SIR
PRODUCTION	TUESDAY Live @11AM	GAURAV SIR
SOM	WEDNESDAY Live @8PM	MUKESH SIR
THERMODYNAMICS	THURSDAY Live @11AM	KANISTH SIR
ENGINEERING MATHEMATICS	FRIDAY Live @11AM	ANANT SIR

😊 $D = \begin{matrix} +0.01 \\ 20\text{mm} \\ -0.02 \end{matrix}$

Tolerances are specified

- (a) To obtain desired fits
- ✓ (b) because it is not possible to manufacture a size exactly
- (c) to obtain higher accuracy
- (d) to have proper allowances



$$25.3 \pm 0.05 \text{ mm}$$

$$* UL = 25.3 + 0.05 = 25.35 \text{ mm}$$

$$* LL = 25.3 - 0.05 = 25.25 \text{ mm}$$

$$* BS = 25.3 \text{ mm}$$

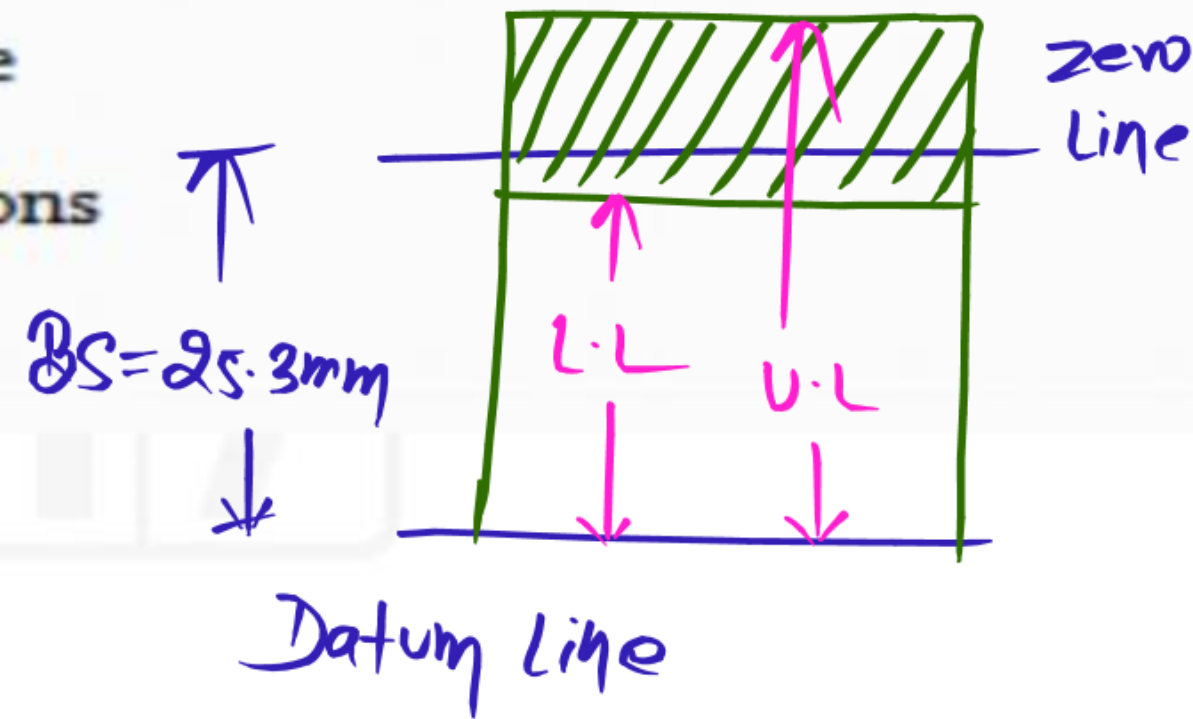
Expressing a dimension as 25.3 ± 0.05 mm is the case of

(a) Unilateral tolerance

(b) Bilateral tolerance

(c) Limiting dimensions

(d) All of the above



 -0.009 Shaft $\phi 35$ mm -0.025

* FD = ?

* Tolerance = ?

A shaft has a dimension, $\phi 35^{+0.009}_{-0.025}$ mm

The respective values of fundamental deviation and tolerance are

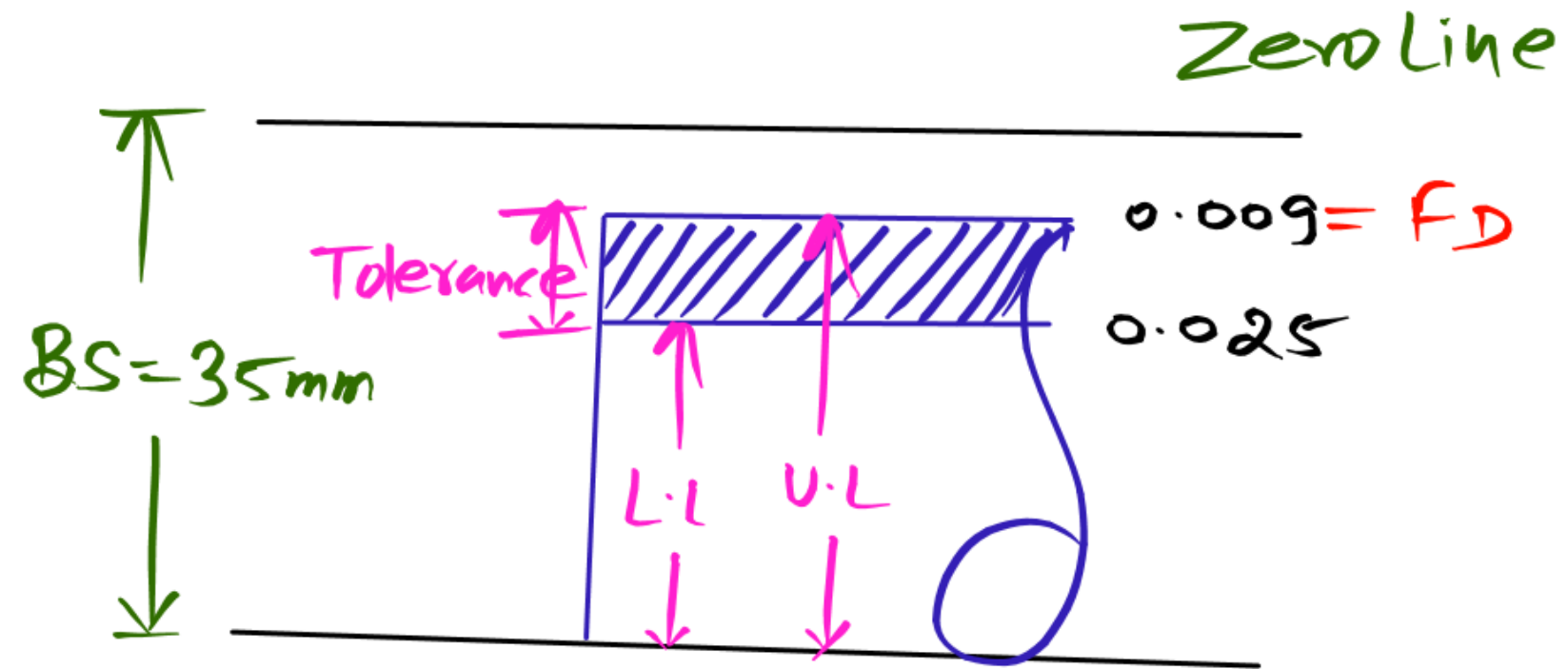
(a) $-0.025, \pm 0.008$ (b) $-0.025, 0.016$ (c) $-0.009, \pm 0.008$ (d) $-0.009, 0.016$

$\phi 35 \text{ mm}$
-0.009
-0.025

* BS = 35 mm

$$UL = 35 - 0.009 = 34.991 \text{ mm}$$

$$LL = 35 - 0.025 = 34.975 \text{ mm}$$



😊 Tolerance = $UL - LL = 34.991 - 34.975$

* Tolerance = 0.016 mm

😊 $F_D \rightarrow$ closest to "Zero Line" $\rightarrow F_D = U_D = -0.009 \text{ mm}$

Datum Line
Unilateral Tolerance



Shaft \rightarrow $25^{+0.040}_{-0.010}$ mm

Hole \rightarrow $25^{+0.020}_{-0.00}$ mm

In an interchangeable assembly, shafts of size

$25.000^{+0.040}_{-0.0100}$ mm mate with holes of size $25.000^{+0.020}_{-0.000}$

mm.

The maximum possible clearance in the assembly will be

- | | |
|----------------|----------------|
| (a) 10 microns | (b) 20 microns |
| (c) 30 microns | (d) 60 microns |

Shaft $25 \begin{matrix} +0.040 \\ -0.010 \end{matrix}$ mm

* UL = 25.040 mm

* LL = 24.99 mm

Hole $25 \begin{matrix} +0.020 \\ -0.00 \end{matrix}$ mm

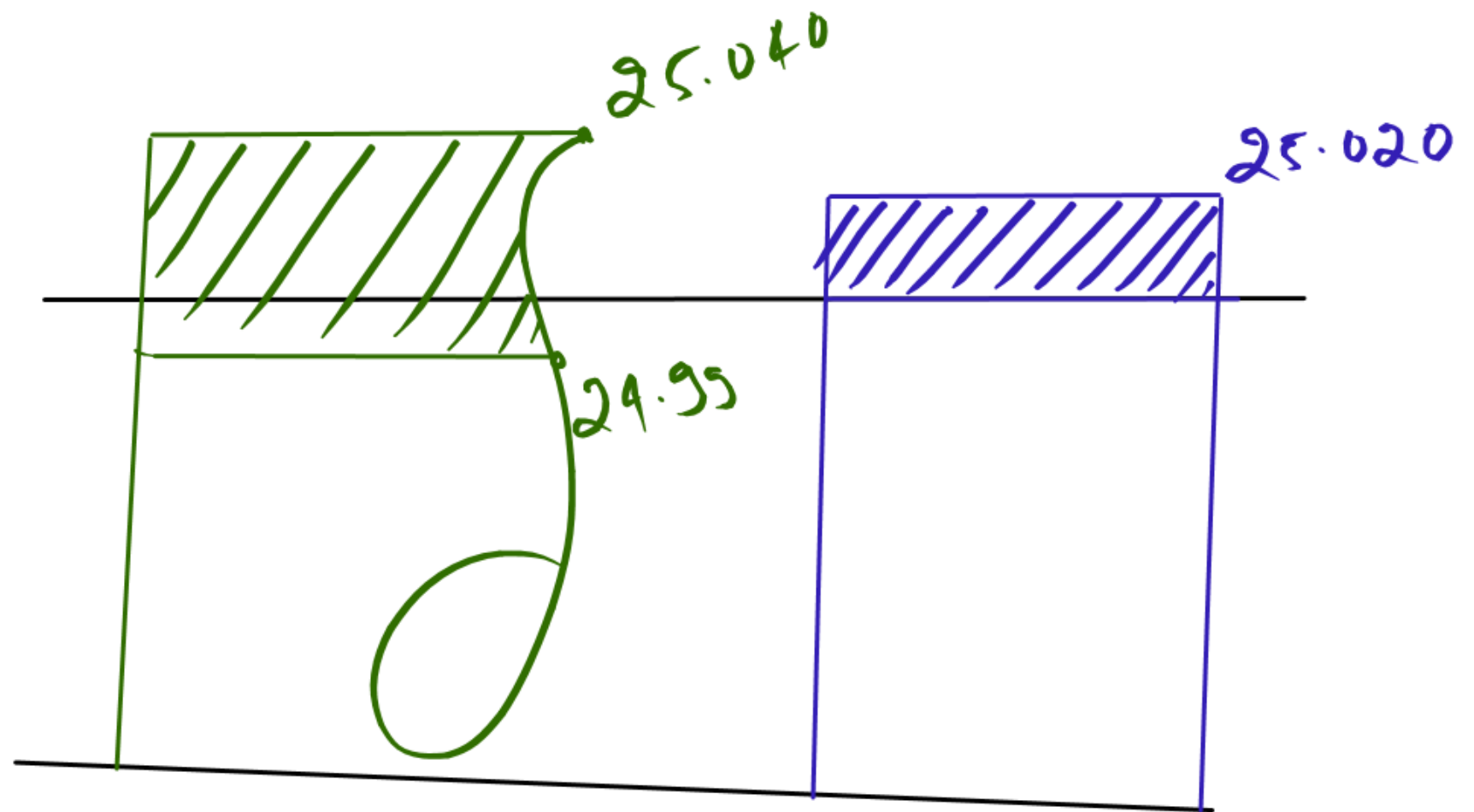
* UL = 25.020 mm

* LL = 25 mm

* Max C = UL of Hole - LL of Shaft

= 25.020 - 24.990 = 0.030 mm = 30 μm

BS = 25 mm



In order to have interference fit, it is essential that the lower limit of the shaft should be

- (a) Greater than the upper limit of the hole
- (b) Lesser than the upper limit of the hole
- (c) Greater than the lower limit of the hole
- (d) Lesser than the lower limit of the hole

Interference fit joints are provided for:

- (a) Assembling bush bearing in housing
- (b) Mounting heavy duty gears on shafts
- (c) Mounting pulley on shafts
- (d) Assembly of flywheels on shafts

Hole $\phi 9^{+0.015}_0$ mm

Shaft $\phi 9^{+0.010}_{+0.001}$ mm

* Hole \rightarrow $UL = 9.015$ mm
 \rightarrow $LL = 9$ mm

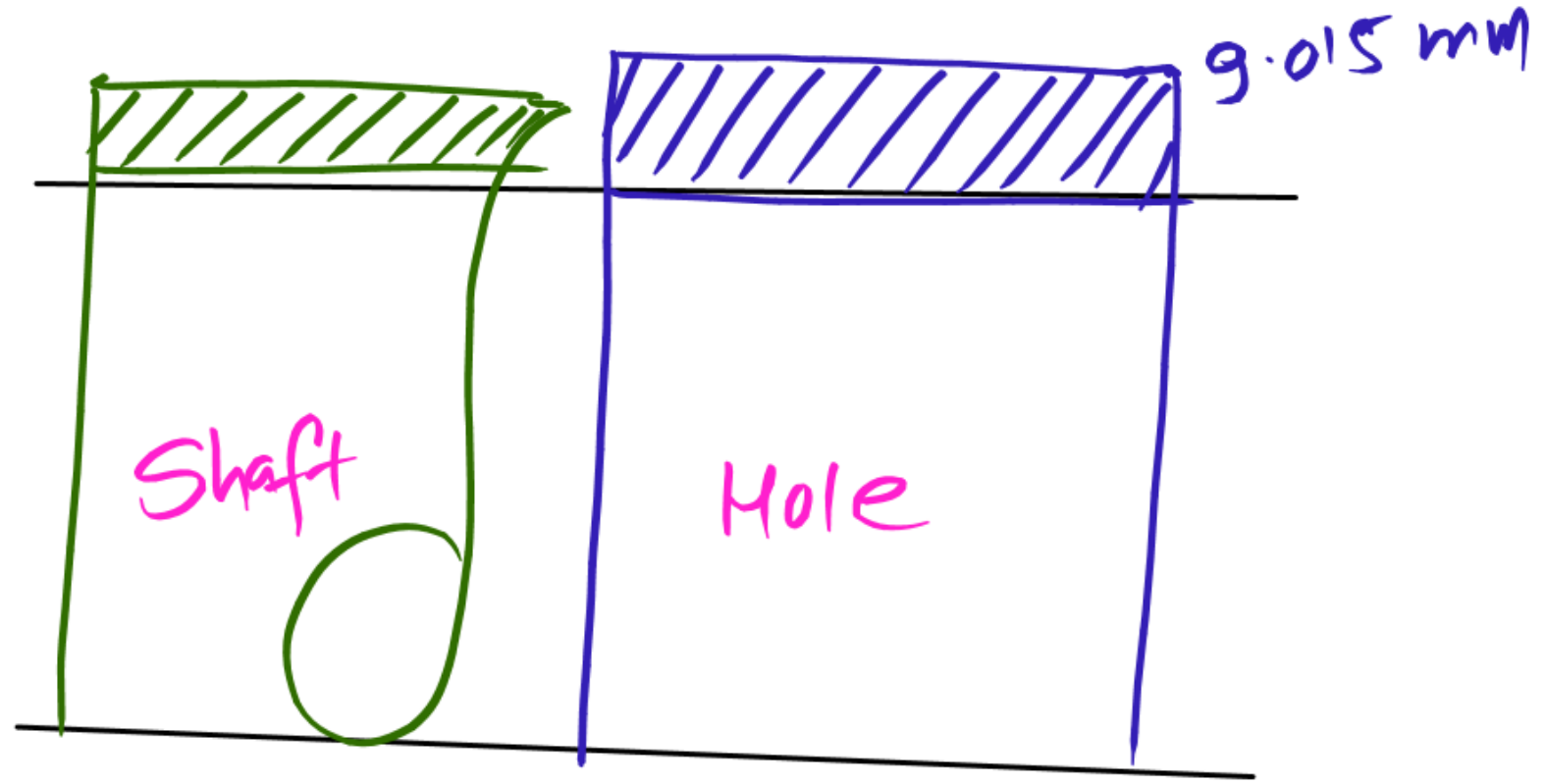
* Shaft \rightarrow $UL = 9.010$ mm
 \rightarrow $LL = 9.001$ mm

A hole is of dimension $\phi 9^{+0.015}_0$ mm. The

corresponding shaft is of dimension $\phi 9^{+0.010}_{+0.001}$ mm.
 The resulting assembly has

- (a) loose running fit
- (b) close running fit
- (c) transition fit
- (d) interference fit

$\phi S = 9\text{mm}$



Shaft \rightarrow $25^{+0.04}_{-0.01}$ mm

Hole \rightarrow $25^{+0.03}_{+0.02}$ mm

In an interchangeable assembly, shafts of size $25^{+0.04}_{-0.01}$

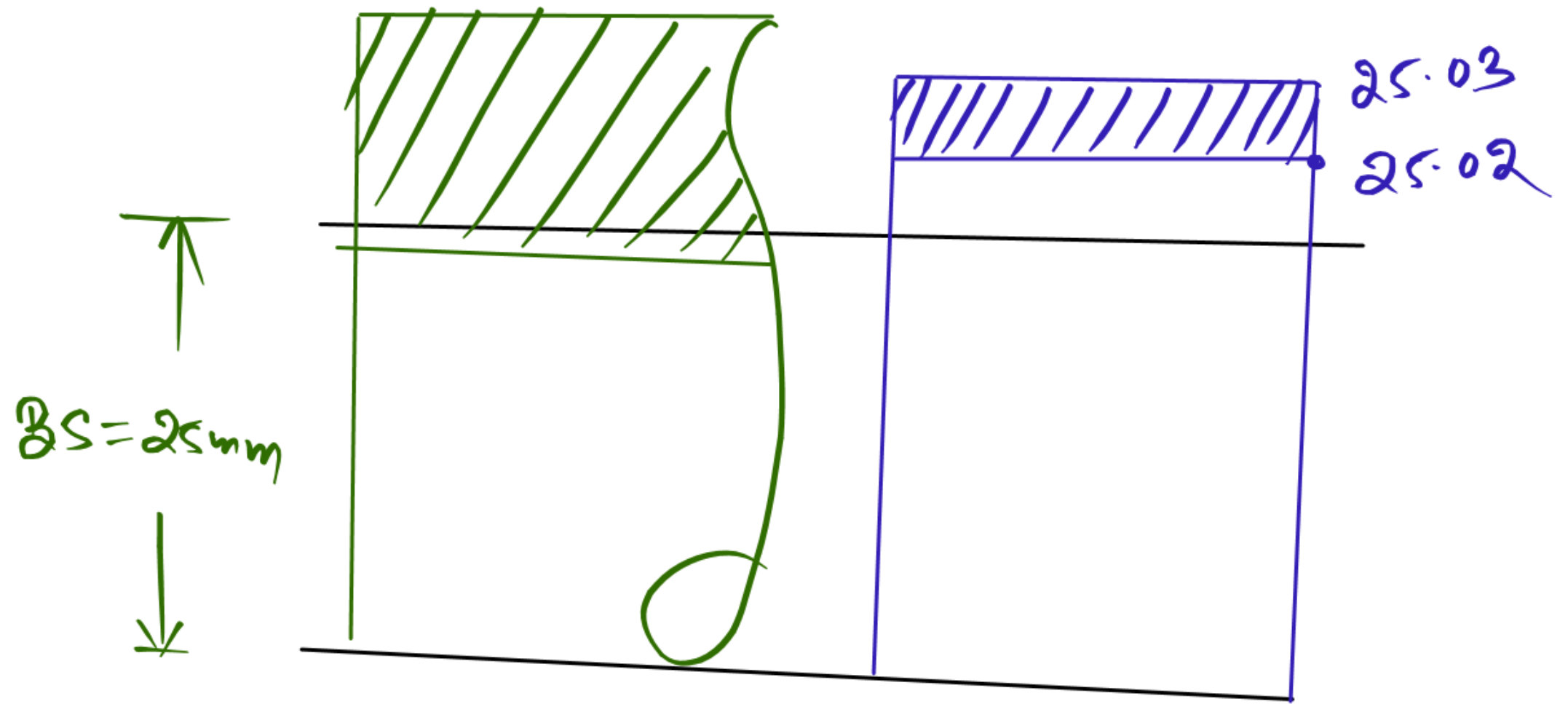
mm mate with holes of size $25^{+0.03}_{+0.02}$ mm.

The maximum interference (in microns) in the assembly is

- (a) 40 (b) 30 ✓ (c) 20 (d) 10

Shaft $\rightarrow 25 \begin{matrix} +0.04 \\ -0.01 \end{matrix} \text{ mm}$
* UL = 25.04 mm
* LL = 24.99 mm

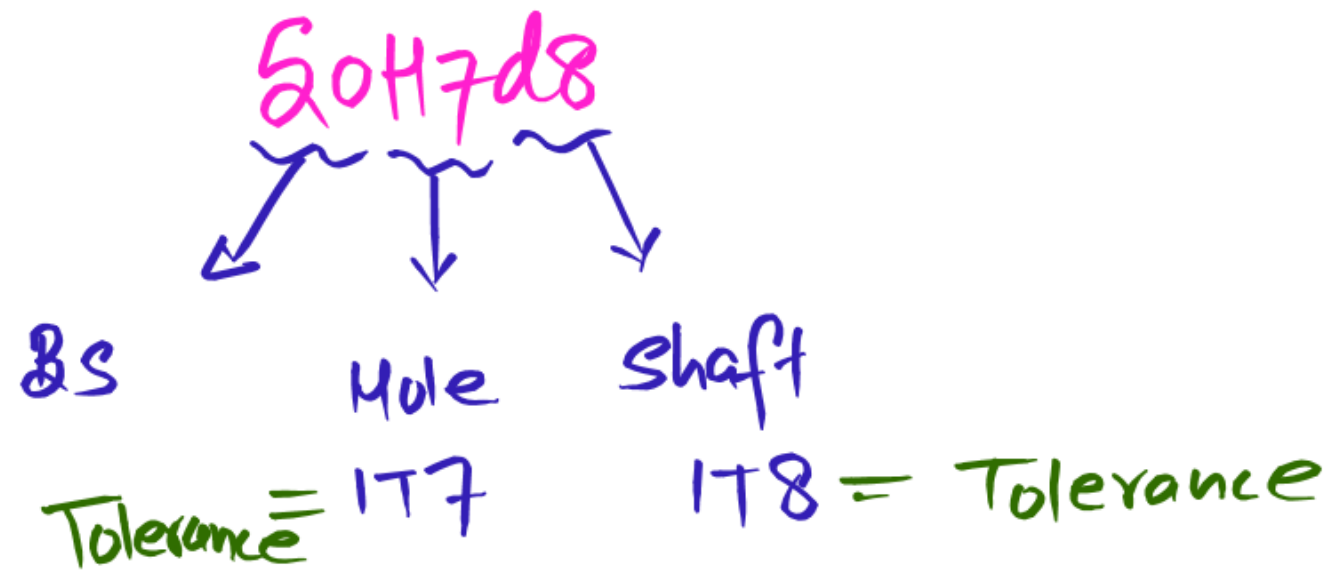
Hole $\rightarrow 25 \begin{matrix} +0.03 \\ +0.02 \end{matrix} \text{ mm}$
* UL = 25.03 mm
* LL = 25.02 mm



Max I = UL of Shaft - LL of Hole

$$25.04 - 25.02 = 0.020 \text{ mm} = 20 \mu\text{m}$$





😊 BS = 50mm
 Tolerance of Hole = IT7
 Tolerance of Shaft = IT8

A shaft and hole pair is designated as 50H7d8.

This assembly constitutes

(a) Interference fit

(b) Transition fit

✓ (c) Clearance fit

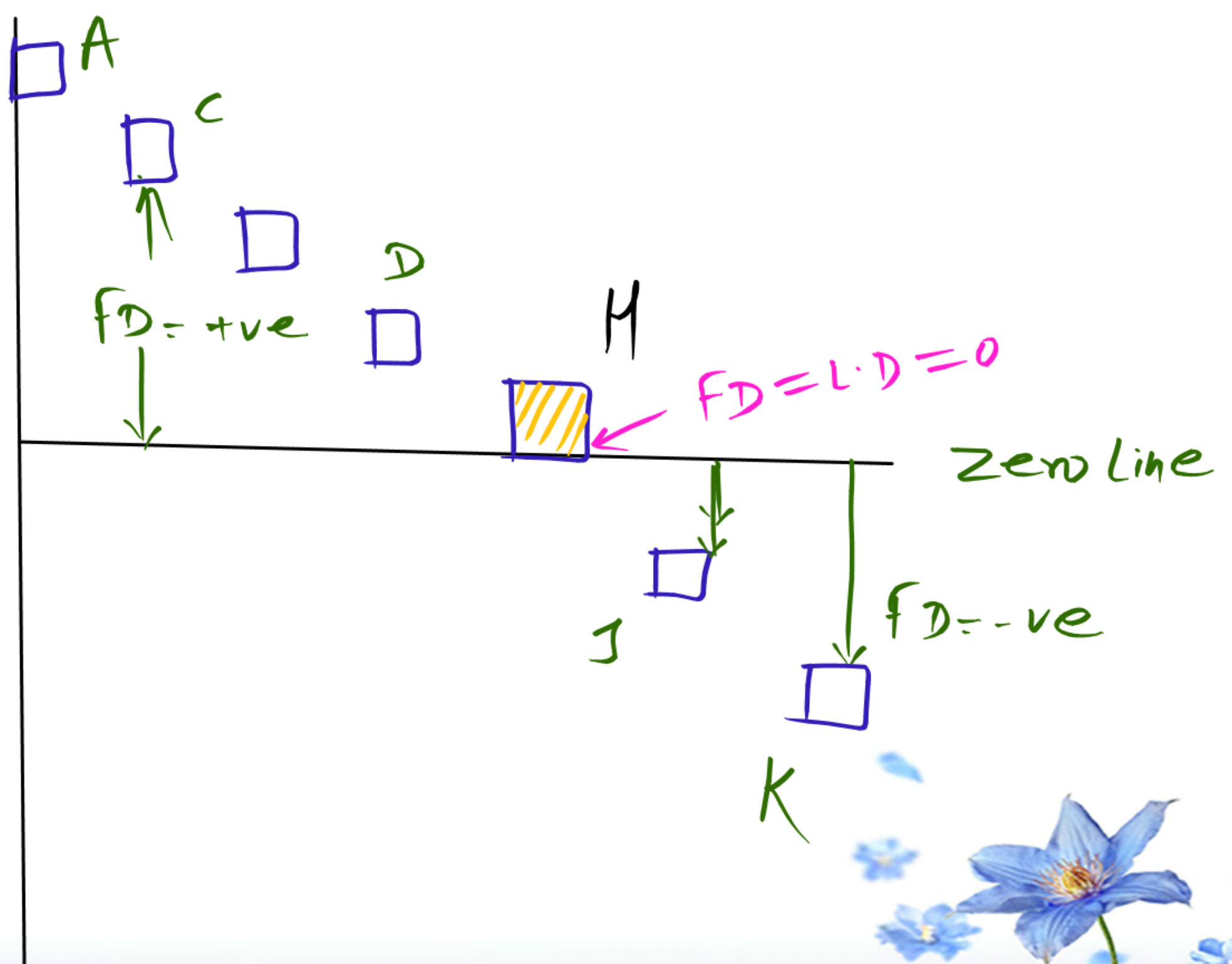
(d) None of the above

* A → Hole

Hole Basis system



* $LD = FD = 0$

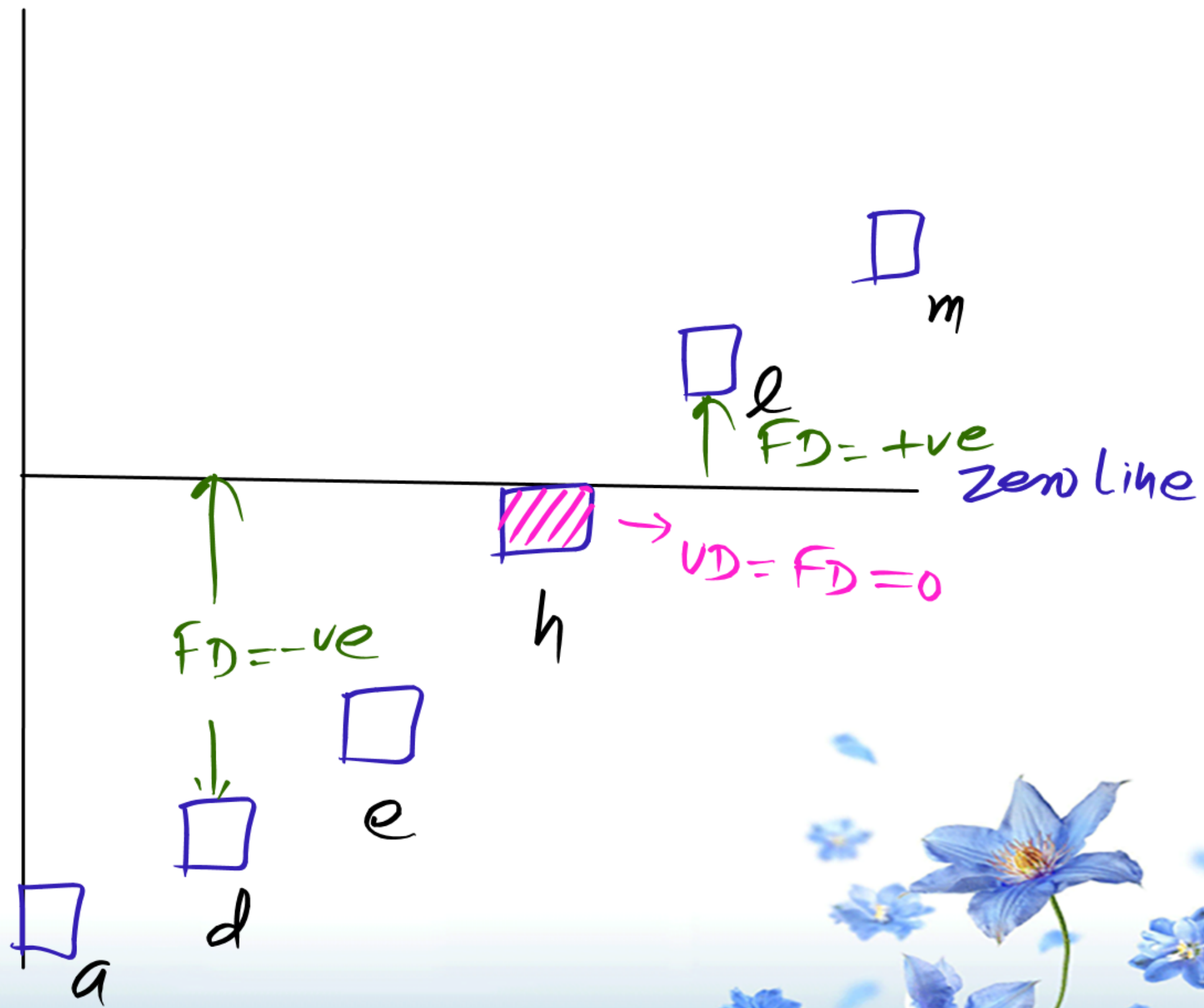


a, b → Shaft

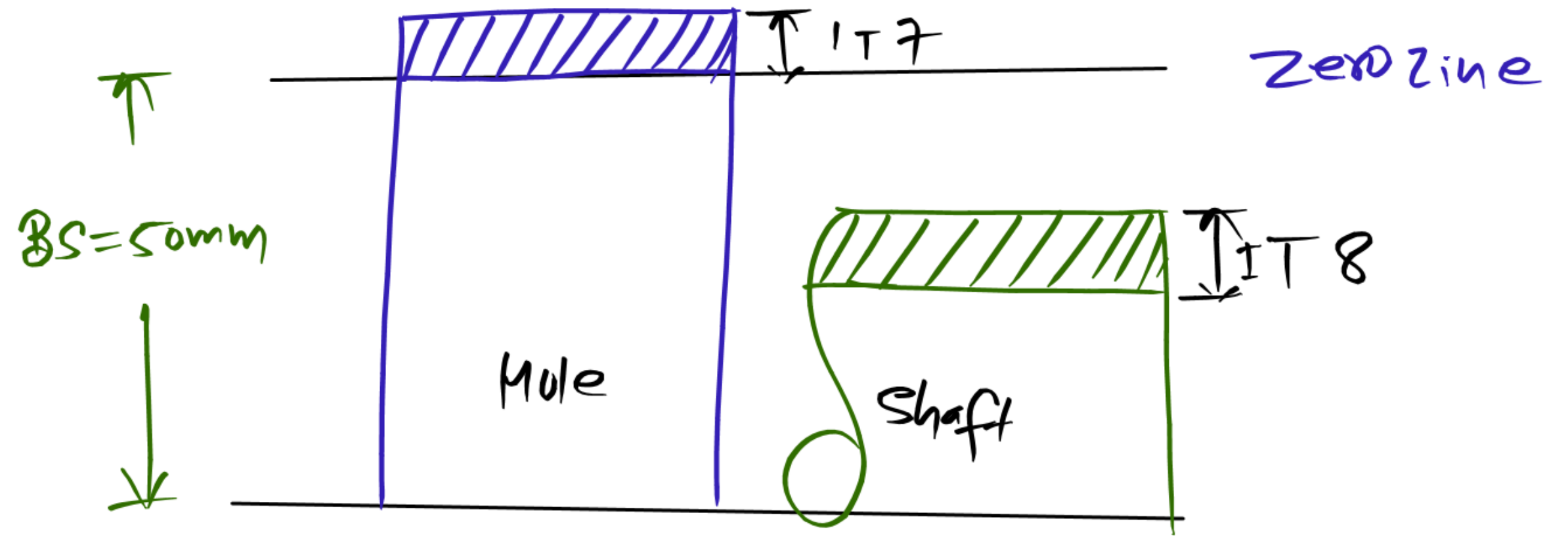
Shaft Basis system



$$VD = FD = 0$$



50H7d8



* clearance fit → slide fit, loose fit
Loose Running fit

Which of the following is an interference fit?

(a) Push fit

(b) Running fit

(c) Sliding fit

(d) Shrink fit

* Transition fit → Tight fit, Push fit

* Interference fit → Shrink fit, Heavy drive fit

Consider the following joints:

1. Railway carriage wheel and axle
2. IC engine cylinder and liner

Which of the above joints is/are the result(s) of interference fit?

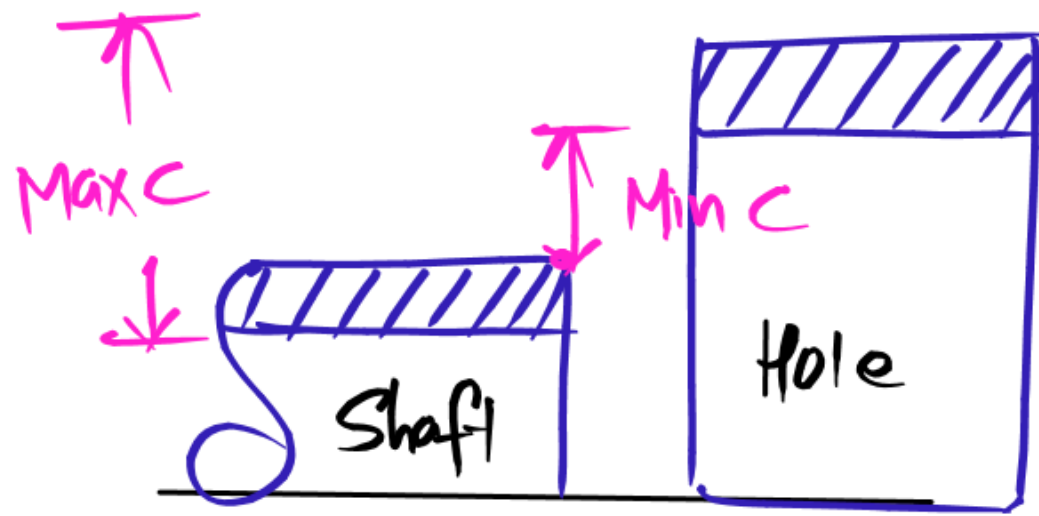
- (a) 1 only
- (b) 2 only
- (c) Neither 1 nor 2
- (d) Both 1 and 2



Allowance = Min C

Allowance in limits and fits refers to

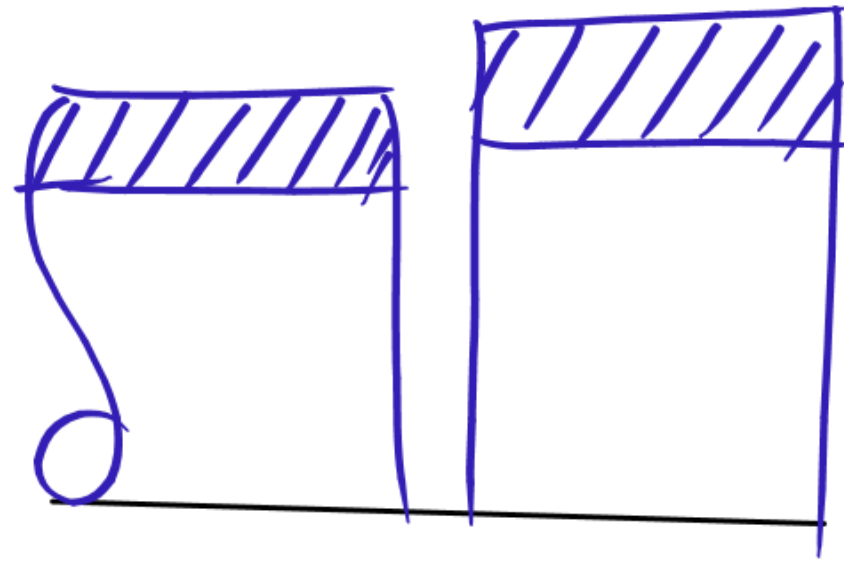
- (a) Maximum clearance between shaft and hole
- ✓ (b) Minimum clearance between shaft and hole
- (c) Difference between maximum and minimum size of hole
- (d) Difference between maximum and minimum size of shaft



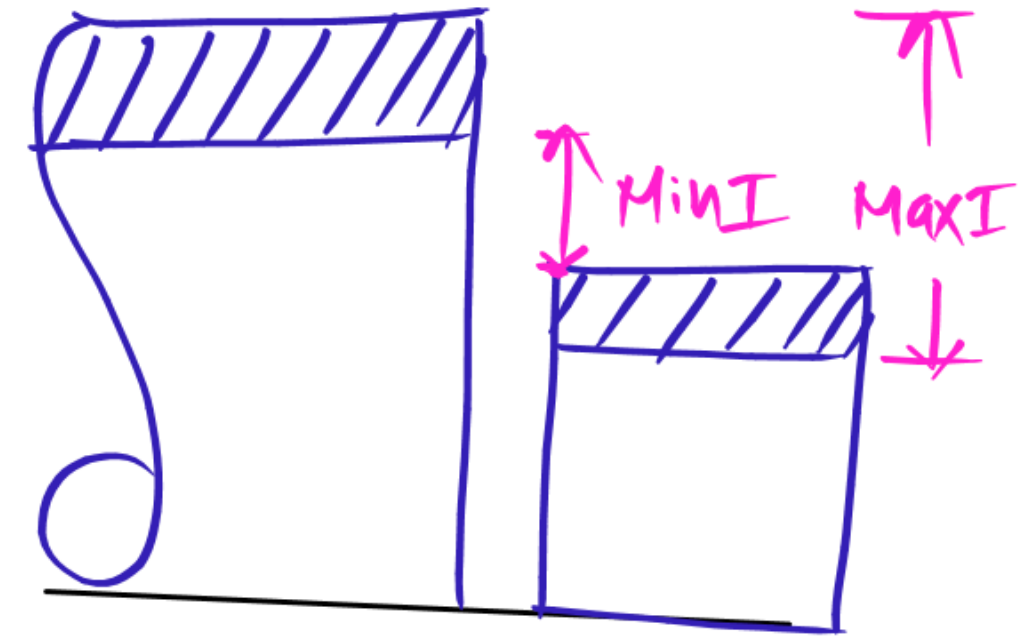
* clearance fit

* $Max C = UL \text{ of Hole} - LL \text{ of shaft}$

* $Min C = UL \text{ of Hole} - UL \text{ of shaft}$



Transition fit



Interference fit

* $Max I = UL \text{ of shaft} - LL \text{ of hole}$

* $Min I = LL \text{ of shaft} - UL \text{ of hole}$





* MinC $\left\{ \begin{array}{l} \rightarrow +ve \rightarrow \text{MinC} \\ \rightarrow -ve \rightarrow \text{MaxI} \end{array} \right.$

$$* \text{MinC} = -\text{MaxI}$$

$$* \text{Allowance} = \text{MinC}$$



Dimension of the hole is $50 \begin{smallmatrix} +0.02 \\ -0.00 \end{smallmatrix} \text{ mm}$

and shaft is $50 \begin{smallmatrix} +0.02 \\ +0.00 \end{smallmatrix} \text{ mm}$.

The minimum clearance is

(a) 0.02 mm

(b) 0.00 mm

(c) -0.02 mm

(d) 0.01 mm



Hole Basis $\rightarrow H$

$$* LD = FD = 0$$

Shaft Basis $\rightarrow h$

$$* UD = FD = 0$$

Basic shaft and basic hole are those whose upper deviations and lower deviations respectively are

(a) +ve, -ve

(b) -ve, +ve

(c) Zero, Zero

(d) None of the above

Which one of the following is not correct in hole basis system of fits?

- (a) The hole size is kept constant.
- (b) The basic size of the hole is taken as the low limit of size of the hole.
- (c) The actual size of a hole that is within the tolerance limits always less than the basic size.
- (d) The high limit of the size of the hole and the two limits of size of the shaft are selected to give desired fit.

25 H7/g8

For the given assembly: 25 H7/g8, match Group A with Group B

Group A	Group B
P. H	I. Shaft Type
Q. IT8	II. Hole Type
R. IT7	III. Hole Tolerance Grade
S. g	IV. Shaft Tolerance Grade

	P	Q	R	S		P	Q	R	S
(a)	I	III	IV	II	(b)	I	IV	III	II
(c)	II	III	IV	I	(d)	II	IV	III	I

$\phi 50 H8/p8$

* BS = 50mm

* Hole Tolerance = IT8

* Shaft Tolerance = IT8

Consider the following statements:

A nomenclature $\phi 50 H8/p8$ denotes that

1. ~~X~~ Hole diameter is 50 mm.
2. ~~X~~ It is a shaft base system. $\rightarrow h8$
3. ~~X~~ 8 indicates fundamental deviation.

Which of the statements given above is/are incorrect?

- (a) 1, 2 and 3
- (b) 1 and 2 only
- (c) 1 and 3 only
- (d) 3 only

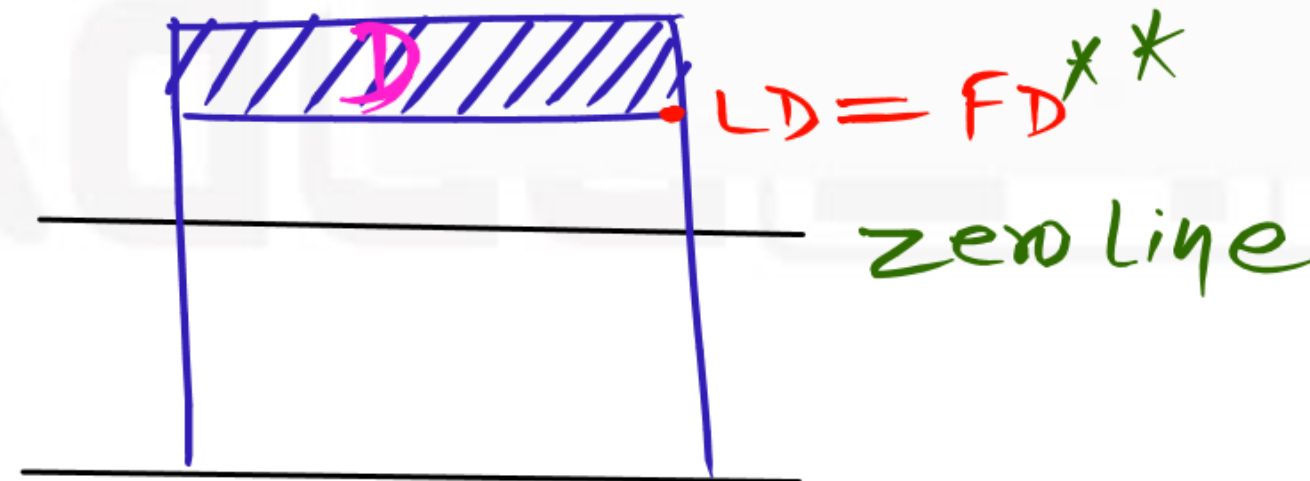
25 D 6

* BS = 25 mm

* Tolerance of Hole = IT 6

In the tolerance specification 25 D 6, the letter D represents

- (a) ~~Grade of tolerance~~
- (b) ~~Upper deviation~~
- (c) ✓ Lower deviation
- (d) ~~Type of fit~~



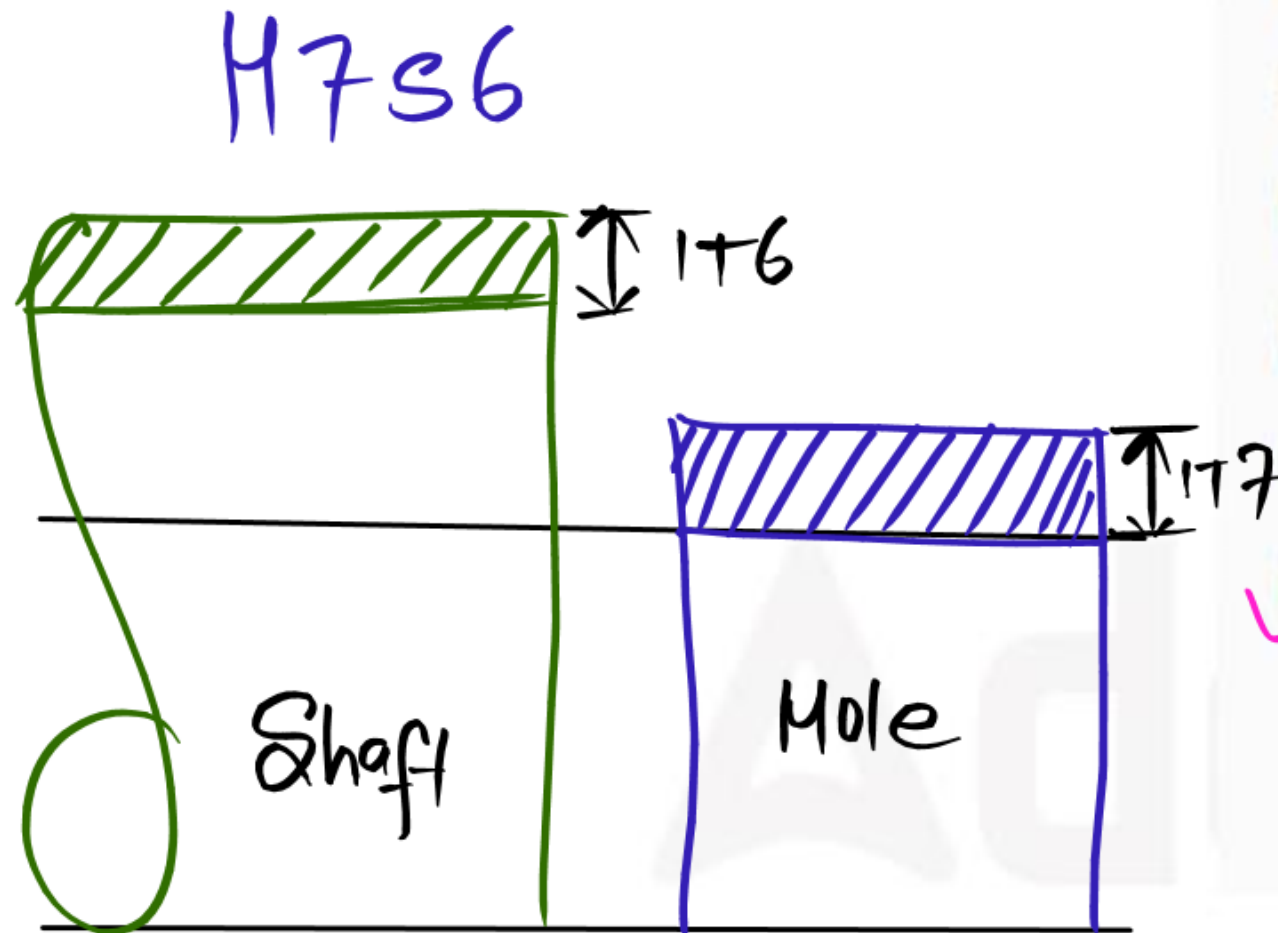
The dimensional limits on a shaft of $25h7$ are

- (a) 25.000, 25.021 mm
- (b) 25.000, 24.979 mm
- (c) 25.000, 25.007 mm
- (d) 25.000, 24.993 mm

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The fit on a hole-shaft system is specified as $H7/s6$. The type of fit is

- (a) Clearance fit
- (b) Running fit (sliding fit)
- (c) Push fit (transition fit)
- (d) Force fit (interference fit)





Gauge



for Inspection OR check

Plug gauges are used to

- ~~(a) Measure the diameter of the workpieces~~
- ~~(b) Measure the diameter of the holes in the workpieces~~
- ✓ (c) Check the diameter of the holes in the workpieces
- ~~(d) Check the length of holes in the workpieces~~

* Plug Gauge → Inspection of Hole

* Ring, Gap, Snap Gauge → Inspection of Shaft

Which one of the following tolerances set on inner diameter and outer diameter respectively of headed jig bush for press fit is correct?

- (a) $G_7 h_6$ (b) $F_7 n_6$
(c) $H_7 h_6$ (d) $F_7 j_6$



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